

Level 3 Assessment as per API 579-1/ASME FFS-1 for Pressure Vessel General Metal Loss by using FEA Techniques.

By

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1. Introduction:

The Fitness-For-Service assessment for pressure vessel during its service subject to general metal loss or local metal loss resulting from corrosion, erosion, or combined condition. The FEA assessment can be used to ensure the integrity of a pressure retaining item for its continual safe operation as Level 1 and Level 2 assessment does not meet the criterion. In this example the thickness average approach has been chosen to provide better conservative result and assumed that the general metal loss occurred inside surface of pressure vessels based on point thickness reading and internal visual inspection. Thus If a more rigorous analysis is performed, no need for re-rate and alteration, saving repair cost or replacement cost and down time.

2. Mechanical Design for Condensate Recovery Drum as per ASME SEC.VIII, DIV.-1.ED 2004

Vessel Data

Material	SA 516 Gr 60 2004 Ed
Design Condition	4.35 Mpa
Design Temp	60 °C
Allow able stress for Temp	138 Mpa
Inside Diameter	1000 mm
Inside radius	500 mm
Nominal Thickness	20 mm
Future Corrosion Allowance	2 mm
Weld Joint Efficiency	1.0
Tangential distance (TL)	3000 mm

Internal corrosion (away from discontinuity) on the cylindrical shell of Condensate Recovery Drum has been found during off stream inspection. Details regarding the Condensate Recovery Drum and inspection data are given below. Determine if the Condensate Recovery Drum is suitable for continued operation.

Inspection Data

Based on a visual inspection, the corrosion loss is characterized as general, and point thickness readings has taken in 15 locations at shell location (see table E2-1)

Table E2-1

Location	Thickness Reading <i>t</i>, mm
1	17
2	16
3	15
4	17
5	14
6	16
7	15
8	17
9	18
10	17
11	15
12	17
13	17
14	18
15	18

3.0 Level 1 Assessment

(Assessment for internal pressure per paragraph - 4.4.2)

STEPS 1 - Use the point thickness readings Table E2.1 shown above; and determine the minimum measured thickness, $t_{min.}$, the average measured thickness, t_{am} , and the Variation, COV.

Where,

COV -Coefficient of Variation.

LOSS -Uniform metal loss away from the damage area at the time of the inspection. *LOSS*

t_{am} . -Average measured wall thickness of the component based on the point thickness readings (PTR) measured at the time of the inspection.

$t_{min.}$ -minimum required wall thickness of the component.

Table E3-1

Location	Thickness Reading $t_{rd,i} = 1 \text{ to } N$	$t_{rd,i} - t_{am}$	$(t_{rd,i} - t_{am})^2$
1	17	0.5333	0.2844
2	16	-0.4667	0.2178
3	15	-1.4667	2.1511
4	17	0.5333	0.2844
5	14	-2.4667	6.0844
6	16	-0.0667	0.0044
7	15	-1.4667	2.1511
8	17	0.5333	0.2844
9	18	1.5333	2.3511
10	17	0.9333	0.8710
11	15	-1.4667	2.1511
12	17	0.5333	0.2844
13	17	0.5333	0.2844
14	18	1.5333	2.3511
15	18	1.5333	2.3511
	$t_{am} = \frac{1}{N} \sum_{i=1}^N t_{rd,i} = 16.4667$	$S = \sum_{i=1}^N (t_{rd,i} - t_{am})^2 =$	22.1066
$COV = \frac{1}{t_{am}} \left[\frac{S}{N-1} \right]^{0.5} = 0.076$		7.6%	Use Average Thickness Method (t_{am})

STEP 2 – The COV equals 7.6%, which is less than 10%; therefore, the average thickness to be used in the calculation is the average thickness of the thickness distribution, or

$$t_{am} = 16.4667$$

$$LOSS = t_{nom.} - t_{am}$$

$$= 20.0 - 16.4667$$

$$= 3.5333 \text{ mm}$$

STEP 3 – Calculate the minimum required thickness

$$t_{cir.} = PR / (SE - 0.6 P)$$

$$= 4.35(500 + 2 + 3.5333) / (138(1.0) - 0.6(4.35))$$

$$= 16.24 \text{ mm}$$

$$\begin{aligned}
 t_{\text{long.}} &= PR / (2SE + 0.4P) \\
 &= 4.35(500+2+3.5333)/2 * 138(1.0)+0.4(4.35) \\
 &= 7.91 \text{ mm}
 \end{aligned}$$

$$\begin{aligned}
 t_{\text{min}} &= \text{Max} \{ t_{\text{cir}}, t_{\text{long.}} \} \\
 &= \text{Max} \{ 16.24, 7.91 \} \\
 &= 16.24 \text{ mm}
 \end{aligned}$$

STEP 4 – Determine if the component is acceptable for continued operation. Perform a Level 1 assessment using Table 4.4 of API 579-1

$$(t_{\text{am}} - \text{FCA}) \geq (t_{\text{min}})$$

$$\begin{aligned}
 (t_{\text{am}} - \text{FCA}) &= (16.4667 - 2.0) \geq 16.24 \\
 &= 14.4667 \geq 16.24 \text{ mm} \quad \text{FALSE}
 \end{aligned}$$

Alternatively, the maximum allowable working pressure MAWP based on the average thickness (t_{am}) can be compared to the design pressure with the design pressure as the criterion.

$$t = t_{\text{am}} - \text{FCA} = 16.4667 - 2.0 = 14.4667$$

$$\begin{aligned}
 \text{MAWP} &= SEt / (R + \text{FCA} + \text{LOSS} + 0.6t) \\
 &= 138 * 1 * 14.4467 / ((500 + 2 + 3.5333 + 0.6 * 14.4467) \\
 &= 3.88 \text{ Mpa}
 \end{aligned}$$

Condition is MAWP > Pd

$$3.88 > 4.35 \quad \text{FALSE}$$

The Level 1 assessment criteria are not satisfied

4.0 Level 2 Assessment

Perform a Level 2 assessment for internal pressure using Table 4.4. Of API 579-1

RSFa (Allowable Remaining Strength Factor) = 0.9 from 2F.2 of API 579

Condition is $(t_{\text{am}} - \text{FCA}) \geq (\text{RSFa} * t_{\text{min}})$

$$(16.4667 - 2.0) \geq (0.9 * 16.24)$$

$$14.4667 \geq 14.616$$

FALSE

Alternatively, the maximum allowable working pressure (MAWP) based on the average thickness (t_{am}) the criterion can be compared to the design pressure with the design pressure as below,

$$t = (t_{am} - FCA) / RSFa = (16.4667 - 2.0) / 0.9 = 16.074$$

$$MAWP = SEt / (R + FCA + LOSS + 0.6t)$$

$$= 138 * 16.074 / (500 + 2 + 3.5333 + 0.6 * 16.074)$$

$$= 4.31 \text{ Mpa}$$

Condition is MAWP > Pd

$$4.31 > 4.35$$

FALSE

The Level 2 assessment criteria are not satisfied.

Now the Possibilities are Level 3 FEA assessments, Re-rate, Repair and Retire.

4.0 Level 3 Assessments.

For Level 3 Assessment, the stress analysis techniques stated in Annex 2D of API 579-1 has been utilized to evaluate regions of local metal loss at Condensate Recovery Drum. The finite element method (ANSYS software) was typically used to compute the stresses in corroded location. The evaluation carried based on a linear stress analysis with acceptability determined using stress categorization and stress linearization as per ASME Section VIII Div.2 part 5 and non-linear stress analysis with acceptability determined using a plastic collapse load. The thickness data was used directly in finite element model of the component. (as per table E.1.1)

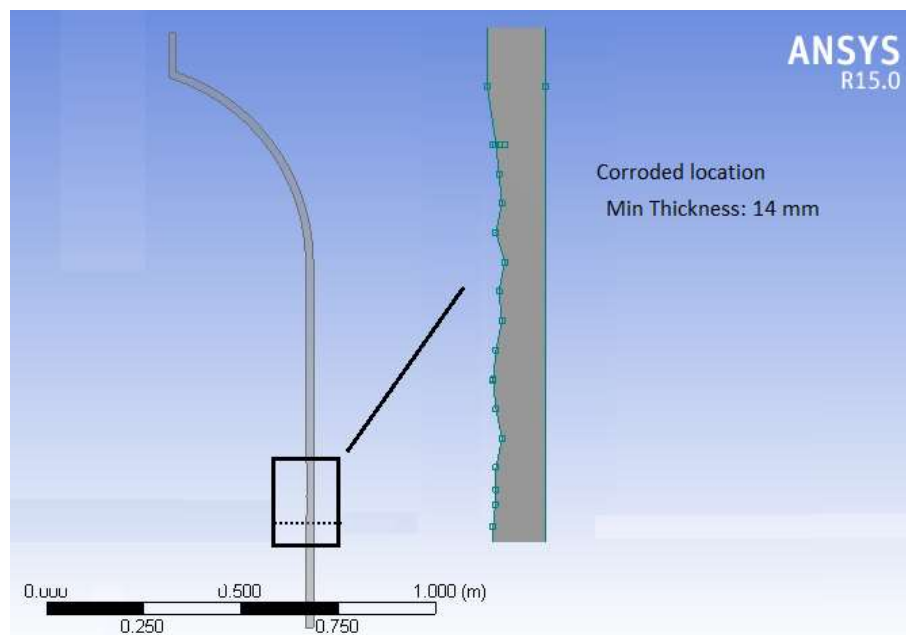


Figure 1 Vessel Shell Geometry with local corroded portion

4.1-Protection against Elastic Stress Analysis and Local Failure

The Elastic Stress Analysis and Local Failure Evaluation were carried at corroded area of shell location for compliance with respect to the elastic stress analysis criteria for plastic collapse and local failure as per guidance provided in ASME Sec VIII Div 2 part 5.2.2 and 5.3.2 .Internal pressure is the only load that was considered with respect its axial thrust pressure in the nozzle location. Other loading conditions are considered negligible.

STEP A – To find out the types of loads acting on the corroded area for fitness for assessment, In general, separate load cases are analyzed to evaluate “load-controlled” loads such as pressure and externally applied reactions due to weight effects and “strain-controlled” loads resulting from thermal gradients (not applicable here) and imposed displacements(not applicable here) The loads to be considered in the assessment shall include, but not be limited to, those given in Table 5.1 of ASME Section VIII Div 2 The load combinations that shall be considered for each loading condition shall include, but not be limited to those given in Table 5.3 of ASME Section VIII Div 2.

The only load considered is internal pressure of 4.35 Mpa and axial thrust pressure of 20.71 Mpa.

- 1) The finite element model has been developed in Ansys Mechanical APDL version 15.

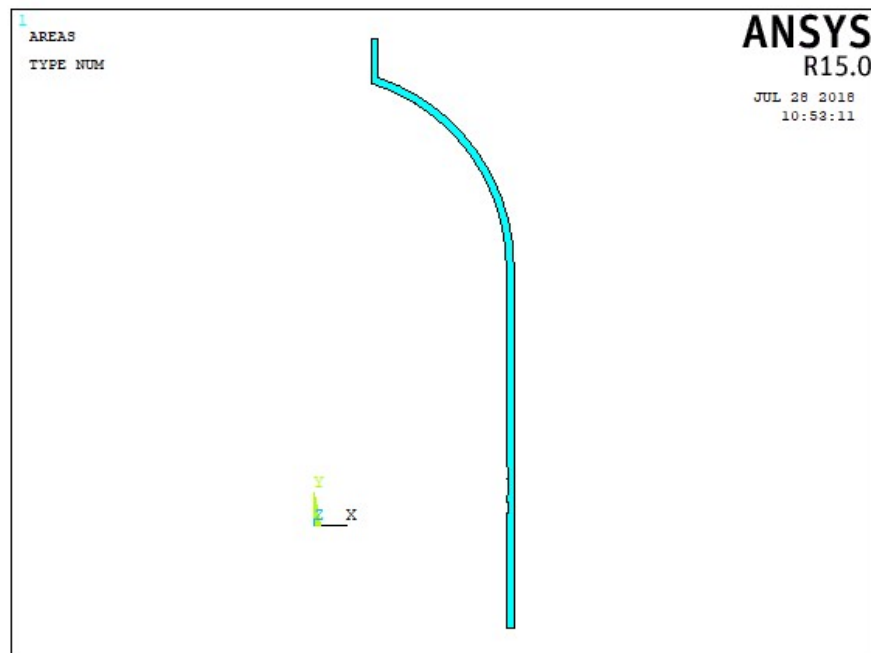


Figure 2 Axisymmetric FE Model

- 2) Due to symmetry in geometry and loading, an axisymmetric solid model is generated. To capture proper membrane behavior in the model the shell was extended a distance of $5\sqrt{Rt}$ below the head to shell transition. The FE model is illustrated in Figure 2. The model was generated with the ANSYS commercial FEA program, version 15.0.

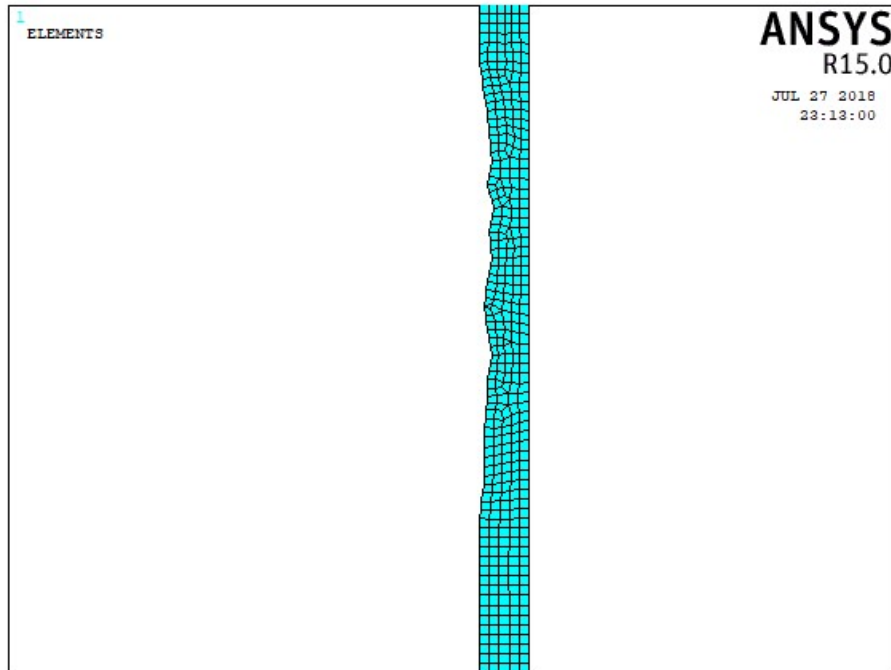


Figure 3 Meshing of Corroded location

3). The Element TYPE 1 PLANE 182 /SOLID /QUAD 4 NODE182 /Full integration was chosen to drive FE solution and generated mesh.

4) Apply the material properties given below to the appropriate components of the model. Material assignments are below,

Material Table

Component	Material	Young's Modulus	Poisson Ratio
Head	SA-516-70N	1.98 E+05	0.3
Shell	SA-516-70N	1.98 E+05	0.3

5) Apply the internal pressure load to the pressure boundaries of the vessel and an appropriate pressure thrust load to the nozzle. Applied the appropriate boundary conditions to the shell edge as per the figure 4.0 below.

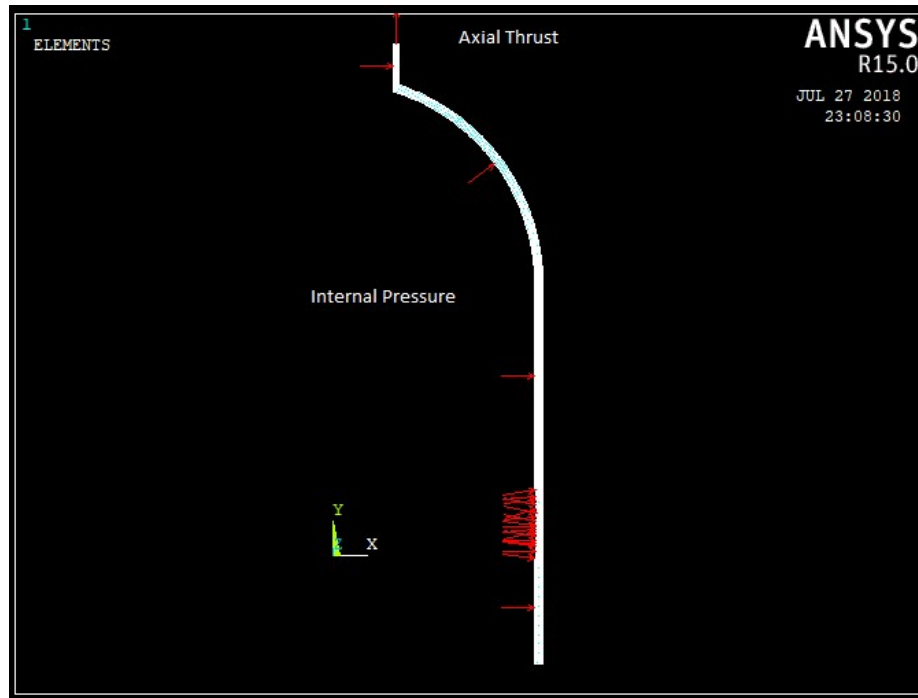


Figure 4. Load and Boundary Conditions for the FE Model

6). Run analysis and reviewed results for Deformation and Von mises equivalent stress below figure 5&6.

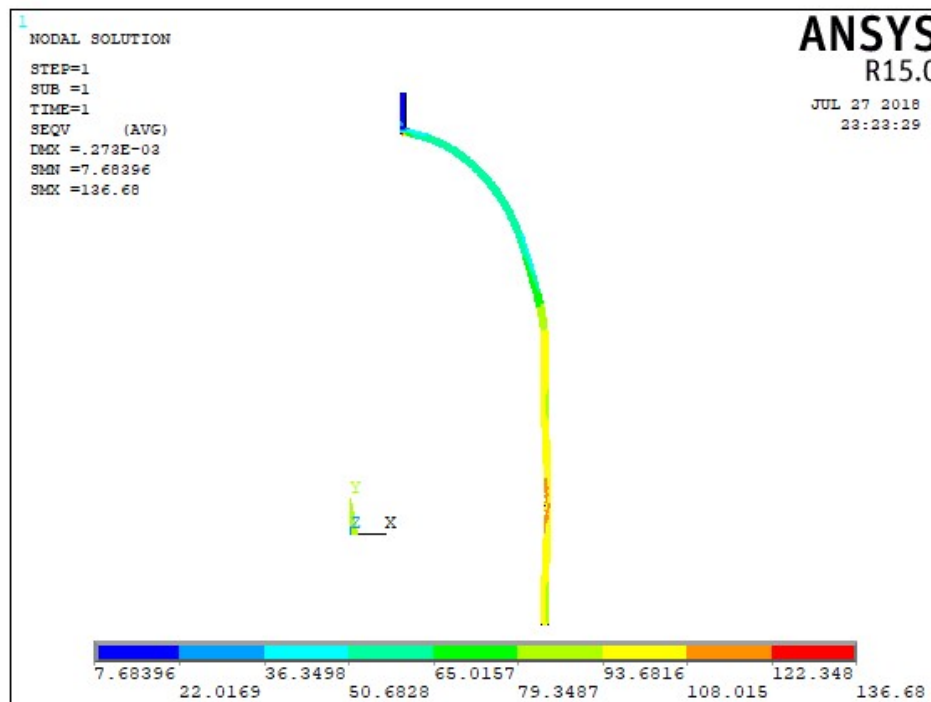


Figure 5. Results of Elastic Analysis, Von Mises Stress in Deformed States

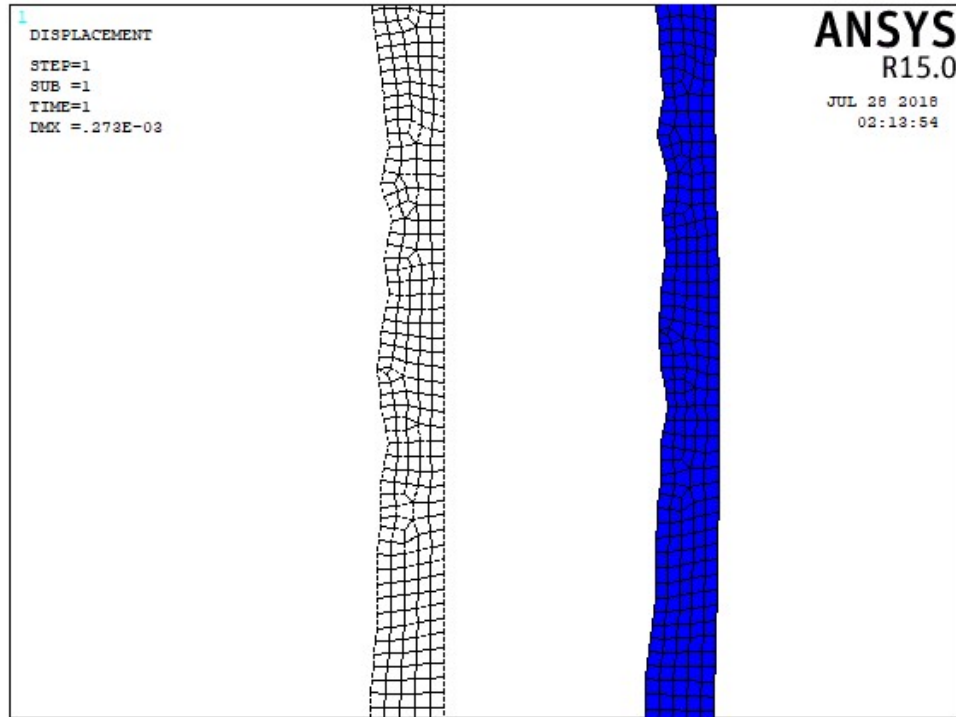


Figure 6. Results of Elastic Analysis, Total Deformation

STEP b) The Through wall stress linearization (Figure 7&8) were conducted at corroded areas in the shell location to provide data for assessment. The resultant von Mises stresses for P_m , P_L , and P_b stress categories are summarized in Table E 4.1

1. Placed proper stress tensor to an appropriate category for a component obtained by using ASME section VIII div. 2 Figure 5.1 and Table 5.6. Note that the equivalent stresses Q and F do not need to be determined to evaluate protection against plastic collapse.

- i) *General primary membrane equivalent stress (P_m) - Average stresses across the thickness at any region away from discontinuities.*
- ii) *Local primary membrane equivalent stress (P_L) - Include the effect of discontinuities.*
- iii) *Primary bending equivalent stress – (P_b) component of primary stress excluding discontinuities*
- iv) *Secondary equivalent stress – (Q) Secondary membrane and bending.*
- v) *Peak Stress (F) - Additional equivalent stress produced by a stress concentration or a thermal stress over and above the nominal ($P + Q$) stress level.*

2. Sum the stress tensors (stresses are added on a component basis) assigned to each equivalent stress category. The final result is a stress tensor representing the effects of all the loads assigned to each equivalent stress category.

3. The P_m equivalent stresses occur away from the head to shell junction, and P_L and $P_L + P_b + Q$ equivalent stresses occur at the junction.

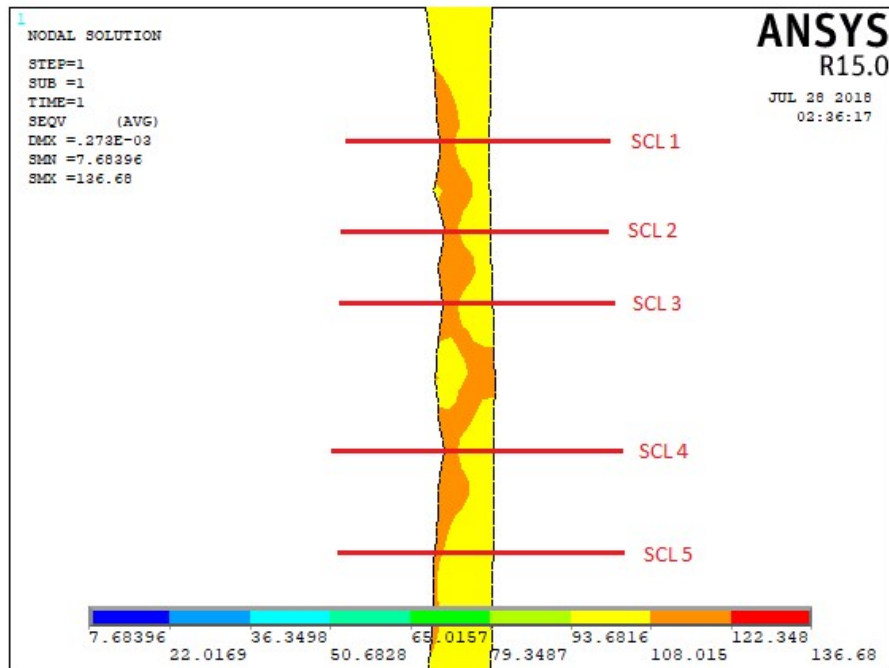


Figure 7. Stress Classification Lines (SCLs) in the corroded location.

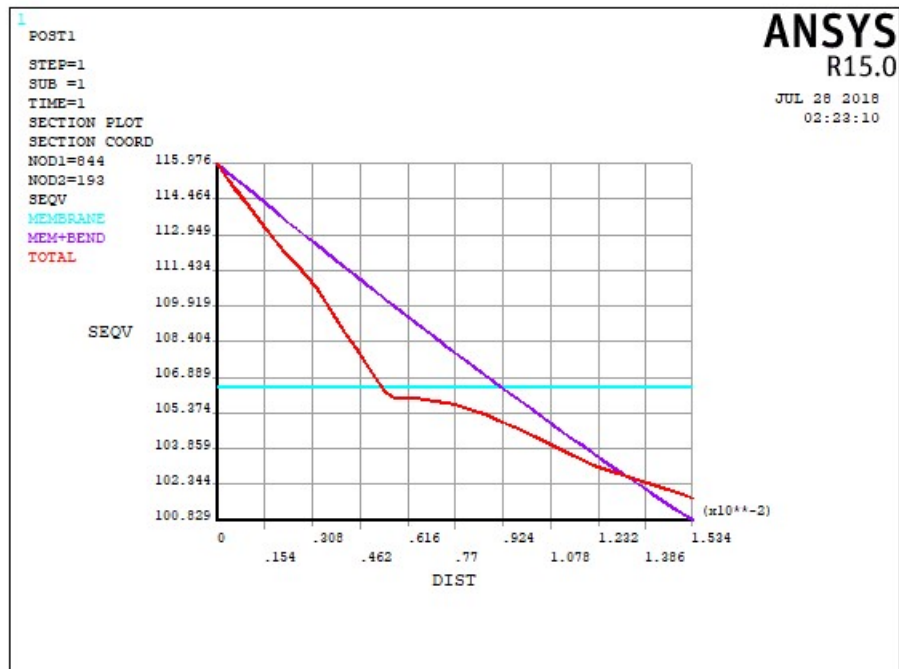


Figure 8. Stress distribution at corroded location

4. To evaluate protection against plastic collapse and local failure, compare the computed equivalent stress to their corresponding allowable values. See Table E4.1 for evaluation results.

(i) For protection against plastic collapse for Elastic Analysis

$$P_m \leq S_m$$

$$P_L \leq 1.5 S_m$$

$$P_L + P_b \leq 1.5 S_m$$

(ii) For protection against local failure (only corroded area).

$$\sigma_1 + \sigma_2 + \sigma_3 \leq 4 S_m$$

Here, σ the algebraic sum of the three linearized primary principal stresses from Design Load Combination.

As per API 579-1, 2D. 3.2, the assessment procedures for the Elastic Analysis of local failure shall be in accordance with VIII-2, Part 5, paragraph 5.3.2 except the acceptance criterion may be $4S/RSF_a$ rather than $4S$ (lenient than ASME Sec.VIII Div.2)

$$4S/RSF_a = 4 * 138 / 0.9$$

$$= 613.33 \text{ Mpa}$$

Table E4.1- Results of the Elastic Analysis and local failure Criterion from ASME Section VIII, Div 2, Part 5

SCL No	Sm	Equivalent Linearized Stresses			Stress Evaluation			Local failure (Triaxial Stress)	Remarks
		Pm	PI	Pb	$P_m \leq S_m$	$P_L \leq 1.5 S_m$	$P_L + P_b \leq 1.5 S_m$	$\sigma_1 + \sigma_2 + \sigma_3 \leq 4 S_m$	
1	138	106.7	NA	NA	Pass	NA	NA	(193.10 ≤ 613.33) Pass	Shell (away from discon.)
2	138	106.8	NA	NA	Pass	NA	NA	(200.25 ≤ 613.33) Pass	Shell (away from discon.)
3	138	107.6	NA	NA	Pass	NA	NA	(193.48 ≤ 613.33) Pass	Shell (away from discon.)
4	138	107.0	NA	NA	Pass	NA	NA	(195.91 ≤ 613.33) Pass	Shell (away from discon.)
5	138	107.1	NA	NA	Pass	NA	NA	(182.37 ≤ 613.33) Pass	Shell (away from discon.)

PRINT LINEARIZED STRESS THROUGH A SECTION DEFINED BY PATH= SCL1 DSYS= 0

RADIUS OF CURVATURE = 508.00

***** POST1 LINEARIZED STRESS LISTING *****

INSIDE NODE = 842 OUTSIDE NODE = 192

LOAD STEP 1 SUBSTEP= 1

TIME= 1.0000 LOAD CASE= 0

** AXISYMMETRIC OPTION ** RHO = 508.00
 THE FOLLOWING X,Y,Z STRESSES ARE IN SECTION COORDINATES.

** MEMBRANE **

SX	SY	SZ	SXY	SYZ	SXZ
-0.2228	70.93	122.4	-0.3525	0.000	0.000
S1	S2	S3	SINT	SEQV	
122.4	70.93	-0.2245	122.7	106.7	

** BENDING ** I=INSIDE C=CENTER O=OUTSIDE

	SX	SY	SZ	SXY	SYZ	SXZ
I	-4.336	11.97	4.856	0.000	0.000	0.000
C	-1.888	0.5809E-01	0.2389E-04	0.000	0.000	0.000
O	0.5598	-11.86	-4.856	0.000	0.000	0.000
	S1	S2	S3	SINT	SEQV	
I	11.97	4.856	-4.336	16.31	14.16	
C	0.5809E-01	0.2389E-04	-1.888	1.946	1.918	
O	0.5598	-4.856	-11.86	12.41	10.78	

** MEMBRANE PLUS BENDING ** I=INSIDE C=CENTER O=OUTSIDE

	SX	SY	SZ	SXY	SYZ	SXZ
I	-4.559	82.90	127.3	-0.3525	0.000	0.000
C	-2.111	70.99	122.4	-0.3525	0.000	0.000
O	0.3370	59.07	117.6	-0.3525	0.000	0.000
	S1	S2	S3	SINT	SEQV	
I	127.3	82.90	-4.560	131.8	116.2	
C	122.4	70.99	-2.112	124.5	108.4	
O	117.6	59.07	0.3349	117.2	101.5	

** PEAK ** I=INSIDE C=CENTER O=OUTSIDE

	SX	SY	SZ	SXY	SYZ	SXZ
I	0.000	7.467	-1.892	3.497	0.000	0.000
C	2.121	-2.474	-0.7519	-0.9831	0.000	0.000
O	-0.1110E-15	3.716	1.067	0.1691	0.000	0.000
	S1	S2	S3	SINT	SEQV	
I	8.848	-1.382	-1.892	10.74	10.49	
C	2.322	-0.7519	-2.675	4.998	4.366	
O	3.723	1.067	-0.7682E-02	3.731	3.326	

** TOTAL ** I=INSIDE C=CENTER O=OUTSIDE

	SX	SY	SZ	SXY	SYZ	SXZ
I	-4.559	90.37	125.4	3.144	0.000	0.000
C	0.1018E-01	68.51	121.7	-1.336	0.000	0.000
O	0.3370	62.79	118.6	-0.1834	0.000	0.000
	S1	S2	S3	SINT	SEQV	TEMP
I	125.4	90.47	-4.663	130.1	116.6	0.000
C	121.7	68.54	-0.1585E-01	121.7	105.7	
O	118.6	62.79	0.3365	118.3	102.5	0.000

PRINT LINEARIZED STRESS THROUGH A SECTION DEFINED BY PATH= SCL2 DSYS= 0

RADIUS OF CURVATURE = 508.00

***** POST1 LINEARIZED STRESS LISTING *****
INSIDE NODE = 848 OUTSIDE NODE = 187

LOAD STEP 1 SUBSTEP= 1
TIME= 1.0000 LOAD CASE= 0

** AXISYMMETRIC OPTION ** RHO = 508.00
THE FOLLOWING X,Y,Z STRESSES ARE IN SECTION COORDINATES.

** MEMBRANE **

SX	SY	SZ	SXY	SYZ	SXZ
1.349	75.11	123.8	0.3591	0.000	0.000
S1	S2	S3	SINT	SEQV	
123.8	75.11	1.348	122.5	106.8	

** BENDING ** I=INSIDE C=CENTER O=OUTSIDE

	SX	SY	SZ	SXY	SYZ	SXZ
I	-5.948	18.42	5.632	0.000	0.000	0.000
C	-3.154	0.8342E-01	0.2587E-04	0.000	0.000	0.000
O	-0.3606	-18.26	-5.632	0.000	0.000	0.000
	S1	S2	S3	SINT	SEQV	
I	18.42	5.632	-5.948	24.37	21.11	
C	0.8342E-01	0.2587E-04	-3.154	3.238	3.197	
O	-0.3606	-5.632	-18.26	17.90	15.93	

** MEMBRANE PLUS BENDING ** I=INSIDE C=CENTER O=OUTSIDE

	SX	SY	SZ	SXY	SYZ	SXZ
I	-4.598	93.53	129.4	0.3591	0.000	0.000
C	-1.805	75.19	123.8	0.3591	0.000	0.000
O	0.9887	56.85	118.2	0.3591	0.000	0.000
	S1	S2	S3	SINT	SEQV	
I	129.4	93.53	-4.600	134.0	120.2	
C	123.8	75.19	-1.806	125.6	109.7	
O	118.2	56.85	0.9864	117.2	101.5	

** PEAK ** I=INSIDE C=CENTER O=OUTSIDE

	SX	SY	SZ	SXY	SYZ	SXZ
I	0.000	14.86	-3.181	-3.988	0.000	0.000
C	4.325	-4.340	-0.6394	0.4178	0.000	0.000
O	0.1110E-15	6.110	1.070	0.5783	0.000	0.000
	S1	S2	S3	SINT	SEQV	
I	15.86	-1.003	-3.181	19.04	18.05	
C	4.346	-0.6394	-4.360	8.705	7.566	
O	6.165	1.070	-0.5424E-01	6.219	5.740	

** TOTAL ** I=INSIDE C=CENTER O=OUTSIDE

	SX	SY	SZ	SXY	SYZ	SXZ
I	-4.598	108.4	126.3	-3.629	0.000	0.000

C	2.521	70.85	123.2	0.7769	0.000	0.000
O	0.9887	62.96	119.3	0.9373	0.000	0.000
	S1	S2	S3	SINT	SEQV	TEMP
I	126.3	108.5	-4.715	131.0	123.1	0.000
C	123.2	70.86	2.512	120.7	104.8	
O	119.3	62.98	0.9746	118.3	102.5	0.000

PRINT LINEARIZED STRESS THROUGH A SECTION DEFINED BY PATH= SCL3 DSYS= 0

RADIUS OF CURVATURE = 508.00

***** POST1 LINEARIZED STRESS LISTING *****

INSIDE NODE = 854 OUTSIDE NODE = 182

LOAD STEP 1 SUBSTEP= 1
TIME= 1.0000 LOAD CASE= 0

** AXISYMMETRIC OPTION ** RHO = 508.00
THE FOLLOWING X,Y,Z STRESSES ARE IN SECTION COORDINATES.

** MEMBRANE **

	SX	SY	SZ	SXY	SYZ	SXZ
	-0.1018	70.79	123.7	1.861	0.000	0.000
	S1	S2	S3	SINT	SEQV	
	123.7	70.84	-0.1507	123.8	107.6	

** BENDING ** I=INSIDE C=CENTER O=OUTSIDE

	SX	SY	SZ	SXY	SYZ	SXZ
I	-3.737	4.045	3.082	0.000	0.000	0.000
C	-1.832	0.1963E-01	0.1517E-04	0.000	0.000	0.000
O	0.7277E-01	-4.005	-3.082	0.000	0.000	0.000
	S1	S2	S3	SINT	SEQV	
I	4.045	3.082	-3.737	7.781	7.347	
C	0.1963E-01	0.1517E-04	-1.832	1.852	1.842	
O	0.7277E-01	-3.082	-4.005	4.078	3.704	

** MEMBRANE PLUS BENDING ** I=INSIDE C=CENTER O=OUTSIDE

	SX	SY	SZ	SXY	SYZ	SXZ
I	-3.839	74.84	126.8	1.861	0.000	0.000
C	-1.934	70.81	123.7	1.861	0.000	0.000
O	-0.2908E-01	66.79	120.6	1.861	0.000	0.000
	S1	S2	S3	SINT	SEQV	
I	126.8	74.88	-3.883	130.6	113.9	
C	123.7	70.86	-1.981	125.7	109.3	
O	120.6	66.84	-0.8088E-01	120.7	104.7	

** PEAK ** I=INSIDE C=CENTER O=OUTSIDE

	SX	SY	SZ	SXY	SYZ	SXZ
I	0.000	8.414	-1.138	3.250	0.000	0.000
C	2.273	-2.268	-0.5571	-0.6180	0.000	0.000
O	0.3469E-17	2.955	0.6652	-0.6249	0.000	0.000

	S1	S2	S3	SINT	SEQV
I	9.523	-1.109	-1.138	10.66	10.65
C	2.356	-0.5571	-2.351	4.707	4.115
O	3.082	0.6652	-0.1267	3.209	2.895

** TOTAL ** I=INSIDE C=CENTER O=OUTSIDE

	SX	SY	SZ	SXY	SYZ	SXZ
I	-3.839	83.25	125.6	5.111	0.000	0.000
C	0.3396	68.54	123.1	1.243	0.000	0.000
O	-0.2908E-01	69.74	121.3	1.236	0.000	0.000

	S1	S2	S3	SINT	SEQV	TEMP
I	125.6	83.55	-4.138	129.8	114.7	0.000
C	123.1	68.57	0.3170	122.8	106.6	
O	121.3	69.77	-0.5098E-01	121.3	105.5	0.000

PRINT LINEARIZED STRESS THROUGH A SECTION DEFINED BY PATH= SCL4 DSYS= 0

RADIUS OF CURVATURE = 508.00

***** POST1 LINEARIZED STRESS LISTING *****

INSIDE NODE = 2 OUTSIDE NODE = 172

LOAD STEP 1 SUBSTEP= 1
TIME= 1.0000 LOAD CASE= 0

** AXISYMMETRIC OPTION ** RHO = 508.00
THE FOLLOWING X,Y,Z STRESSES ARE IN SECTION COORDINATES.

** MEMBRANE **

	SX	SY	SZ	SXY	SYZ	SXZ
	1.220	70.40	124.3	3.239	0.000	0.000

	S1	S2	S3	SINT	SEQV
	124.3	70.55	1.069	123.2	107.0

** BENDING ** I=INSIDE C=CENTER O=OUTSIDE

	SX	SY	SZ	SXY	SYZ	SXZ
I	-5.496	12.36	5.285	0.000	0.000	0.000
C	-3.027	0.5998E-01	0.2602E-04	0.000	0.000	0.000
O	-0.5582	-12.24	-5.285	0.000	0.000	0.000

	S1	S2	S3	SINT	SEQV
I	12.36	5.285	-5.496	17.85	15.57
C	0.5998E-01	0.2602E-04	-3.027	3.087	3.058
O	-0.5582	-5.285	-12.24	11.68	10.17

** MEMBRANE PLUS BENDING ** I=INSIDE C=CENTER O=OUTSIDE

	SX	SY	SZ	SXY	SYZ	SXZ
I	-4.276	82.76	129.5	3.239	0.000	0.000
C	-1.807	70.46	124.3	3.239	0.000	0.000
O	0.6620	58.17	119.0	3.239	0.000	0.000

	S1	S2	S3	SINT	SEQV
--	----	----	----	------	------

I	129.5	82.88	-4.396	133.9	117.8
C	124.3	70.61	-1.952	126.2	109.7
O	119.0	58.35	0.4801	118.5	102.6

** PEAK ** I=INSIDE C=CENTER O=OUTSIDE

	SX	SY	SZ	SXY	SYZ	SXZ
I	-0.8882E-15	13.08	-1.903	0.6734	0.000	0.000
C	3.754	-3.549	-0.7228	0.3267	0.000	0.000
O	0.3331E-15	5.040	1.375	-0.7196	0.000	0.000
	S1	S2	S3	SINT	SEQV	
I	13.12	-0.3457E-01	-1.903	15.02	14.18	
C	3.769	-0.7228	-3.564	7.333	6.404	
O	5.141	1.375	-0.1007	5.242	4.682	

** TOTAL ** I=INSIDE C=CENTER O=OUTSIDE

	SX	SY	SZ	SXY	SYZ	SXZ
I	-4.276	95.84	127.6	3.912	0.000	0.000
C	1.947	66.91	123.5	3.566	0.000	0.000
O	0.6620	63.21	120.3	2.520	0.000	0.000
	S1	S2	S3	SINT	SEQV	TEMP
I	127.6	95.99	-4.429	132.1	119.4	0.000
C	123.5	67.11	1.752	121.8	105.6	

PRINT LINEARIZED STRESS THROUGH A SECTION DEFINED BY PATH= SCL5 DSYS= 0

RADIUS OF CURVATURE = 508.00

***** POST1 LINEARIZED STRESS LISTING *****

INSIDE NODE = 8 OUTSIDE NODE = 167

LOAD STEP 1 SUBSTEP= 1
TIME= 1.0000 LOAD CASE= 0

** AXISYMMETRIC OPTION ** RHO = 508.00
THE FOLLOWING X,Y,Z STRESSES ARE IN SECTION COORDINATES.

** MEMBRANE **

	SX	SY	SZ	SXY	SYZ	SXZ
	-1.573	62.45	121.5	-5.920	0.000	0.000
	S1	S2	S3	SINT	SEQV	
	121.5	62.99	-2.116	123.6	107.1	

** BENDING ** I=INSIDE C=CENTER O=OUTSIDE

	SX	SY	SZ	SXY	SYZ	SXZ
I	-2.048	-2.246	1.440	0.000	0.000	0.000
C	-0.1309	-0.1237E-01	0.8078E-05	0.000	0.000	0.000
O	1.786	2.221	-1.440	0.000	0.000	0.000
	S1	S2	S3	SINT	SEQV	
I	1.440	-2.048	-2.246	3.685	3.591	
C	0.8078E-05	-0.1237E-01	-0.1309	0.1309	0.1252	
O	2.221	1.786	-1.440	3.661	3.464	


```

** MEMBRANE PLUS BENDING **  I=INSIDE C=CENTER O=OUTSIDE
      SX      SY      SZ      SXY      SYZ      SXZ
I  -3.621    60.20    122.9    -5.920    0.000    0.000
C  -1.704    62.44    121.5    -5.920    0.000    0.000
O   0.2132   64.67    120.0    -5.920    0.000    0.000
      S1      S2      S3      SINT     SEQV
I   122.9    60.75    -4.166    127.1    110.1
C   121.5    62.98    -2.246    123.7    107.2
O   120.0    65.21    -0.3260   120.4    104.4

```

```

** PEAK **  I=INSIDE C=CENTER O=OUTSIDE
      SX      SY      SZ      SXY      SYZ      SXZ
I   0.000    1.041    -0.1982   0.7776   0.000    0.000
C -0.5316E-01 -0.2285   -0.1832  -0.8610E-02  0.000    0.000
O -0.1388E-15  0.3495    0.5196E-01  -1.287   0.000    0.000
      S1      S2      S3      SINT     SEQV
I   1.456    -0.1982   -0.4151    1.872    1.773
C -0.5274E-01 -0.1832   -0.2290    0.1762   0.1584
O   1.474    0.5196E-01  -1.124    2.598    2.254

```

```

** TOTAL **  I=INSIDE C=CENTER O=OUTSIDE
      SX      SY      SZ      SXY      SYZ      SXZ
I  -3.621    61.25    122.7    -5.143    0.000    0.000
C  -1.757    62.21    121.3    -5.929    0.000    0.000
O   0.2132   65.02    120.1    -7.207    0.000    0.000
      S1      S2      S3      SINT     SEQV      TEMP
I   122.7    61.65    -4.026    126.7    109.8    0.000
C   121.3    62.75    -2.302    123.6    107.1
O   120.1    65.81    -0.5786   120.7    104.7    0.000

```

4.2-Limit-Load Analysis Method

The Non-linear stress analysis techniques are recommended as per API 579-1/ ASME FSS-1 part 4.4.4 Non Linear assessment is best way of estimate of the acceptable load carrying capacity of the component. The assessment procedures for the Limit-Load Analysis Method shall be in accordance with VIII-2, Part 5, paragraph 5.2.3 except that the load case combinations that incorporate the Allowable Remaining Strength Factor as shown in Table 2D.3 may be used in the assessment and $RSF_a = 0.9$

Table 2D.3 – Load Case Combinations and Load Factors for a Limit Load Analysis

Design Conditions		
Criteria	Required Factored Load Combinations	
Global Criteria	Load Case	Load Combination
	1	$[1.5(P + P_i + D)] \cdot RSF_a$
	2	$[1.5(P + P_i + D + T) + 1.7L + 0.54S_s] \cdot RSF_a$
	3	$[1.3(P + P_i + D) + 1.7S_s + (1.1L \text{ or } 0.54W)] \cdot RSF_a$
	4	$[1.3(P + P_i + D) + 1.1W + 1.1L + 0.54S_s] \cdot RSF_a$
	5	$[1.3(P + P_i + D) + 1.1E + 1.1L + 0.21S_s] \cdot RSF_a$

The conditions for direct assembly have been met. No .emat or .erot files will be produced.

*** WARNING *** CP = 42.562 TIME= 08:54:45
The program chosen initial timestep/load-factor is arbitrary. It is necessary for the user to supply a suitable initial timestep/load-factor through the NSUB or DELTIM command for convergence and overall efficiency.

LOAD STEP OPTIONS

LOAD STEP NUMBER. 1
TIME AT END OF THE LOAD STEP. 1.0000
AUTOMATIC TIME STEPPING ON
INITIAL NUMBER OF SUBSTEPS 1
MAXIMUM NUMBER OF SUBSTEPS 1000
MINIMUM NUMBER OF SUBSTEPS 1
START WITH TIME STEP FROM PREVIOUS SUBSTEP. YES
MAXIMUM NUMBER OF EQUILIBRIUM ITERATIONS. 15
STEP CHANGE BOUNDARY CONDITIONS NO
TERMINATE ANALYSIS IF NOT CONVERGED YES (EXIT)
CONVERGENCE CONTROLS USE DEFAULTS
COPY INTEGRATION POINT VALUES TO NODE YES, FOR ELEMENTS WITH

ACTIVE MAT. NONLINEARITIES
PRINT OUTPUT CONTROLS NO PRINTOUT
DATABASE OUTPUT CONTROLS ALL DATA WRITTEN
FOR THE LAST SUBSTEP

DISP CONVERGENCE VALUE = 0.1511E-01 CRITERION= 0.1792E-04
EQUIL ITER 1 COMPLETED. NEW TRIANG MATRIX. MAX DOF INC= 0.3584E-03
FORCE CONVERGENCE VALUE = 0.4575E-08 CRITERION= 0.1295E-01 <<< CONVERGED
curEqn= 5606 totEqn= 5605 Job CP sec= 44.547
Factor Done= 100% Factor Wall sec= 0.006 rate= 514.2 Mflops
DISP CONVERGENCE VALUE = 0.1449E-10 CRITERION= 0.1792E-04 <<< CONVERGED
EQUIL ITER 2 COMPLETED. NEW TRIANG MATRIX. MAX DOF INC = 0.3769E-12

>>> SOLUTION CONVERGED AFTER EQUILIBRIUM ITERATION 2

Step 5 .The results of the analysis are shown in above, convergence was achieved at the factored load condition; therefore the vessel passes limit load analysis for this load case.

4.3- Elastic-Plastic Analysis

The Non-linear stress analysis techniques are recommended as per API 579-1/ ASME FSS-1 part 4.4.4 Non Linear assessment is best way of estimate of the acceptable load carrying capacity of the component. The assessment procedures for Elastic-Plastic Analysis shall be in accordance with VIII-2, Part 5, paragraph 5.2.4 except that the load case combinations that incorporate the Allowable Remaining Strength Factor as shown in Table 2D.5 may be used in the assessment and RSFa = 0.9

Table 2D.5 – Values of β for an FFS Assessment Based on Elastic-Plastic Analysis

Construction Code	β
PD 5500	$2.35 \cdot RSF_a (1)$
EN 13345	$2.4 \cdot RSF_a (1)$
ASME Section VIII, Division 2, 2007 Edition and later	$2.4 \cdot RSF_a (1)$
ASME Section VIII, Division 2, prior to the 2007 Edition	$3.0 \cdot RSF_a (1)$
ASME Section VIII, Division 1, 1999 Edition and later	$3.5 \cdot RSF_a (1)$
ASME Section VIII, Division 1, prior to the 1999 Edition	$4.0 \cdot RSF_a (1)$

STEP 1 – Develop a numerical model of the component including all relevant geometry characteristics. The model used for the analysis shall be selected to accurately represent the component geometry, boundary conditions, and applied loads. The same model was used as in see Figure 1 & 2.

STEP 2 – Define all relevant loads and applicable load cases. The only load to be considered is internal pressure and associated thrust pressure.

$$2.4 (P) \times RSFa = 2.4*(4.35)*0.9 = 9.39 \text{ Mpa and axial trust on nozzle} = 44.74 \text{ Mpa}$$

STEP 3 – An elastic-perfectly plastic model was utilized and von Mises yield function and associated flow rule was utilized. The effects of non-linear geometry are considered in the analysis.

STEP 4- Perform an elastic-plastic analysis. **If convergence is achieved**, the component is stable under the applied loads for this load case. Otherwise, the component configuration does not meet the criterion or

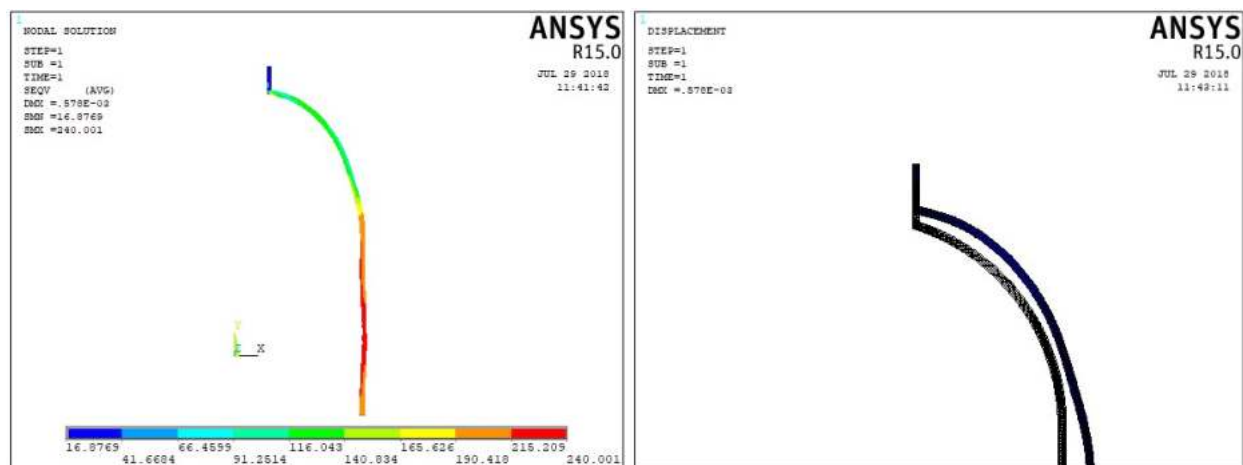


Figure 11&12 - Results of the Limit load Analysis at Factored Load of 9.39 Mpa

SOLUTION OPTIONS

```
PROBLEM DIMENSIONALITY. . . . .AXISYMMETRIC
DEGREES OF FREEDOM. . . . . UX   UY
ANALYSIS TYPE . . . . .STATIC (STEADY-STATE)
PLASTIC MATERIAL PROPERTIES INCLUDED. . . . .YES
NEWTON-RAPHSON OPTION . . . . .PROGRAM CHOSEN
GLOBALLY ASSEMBLED MATRIX . . . . .SYMMETRIC
```

```
*** NOTE ***                      CP =      86.656    TIME= 11:17:34
Present time 0 is less than or equal to the previous time.  Time will
default to 1.
```

```
*** NOTE ***                      CP =      86.656    TIME= 11:17:34
Results printout suppressed for interactive execute.
```

```
*** NOTE ***                      CP =      86.656    TIME= 11:17:34
This nonlinear analysis defaults to using the full Newton-Raphson
solution procedure.  This can be modified using the NROPT command.
```

```
SOLCONTROL,ON uses sparse matrix direct solver
```

```
*** NOTE ***                      CP =      86.703    TIME= 11:17:34
```

The conditions for direct assembly have been met. No .emat or .erot files will be produced.

*** WARNING ***

CP = 86.703 TIME= 11:17:35

The program chosen initial timestep/load-factor is arbitrary. It is necessary for the user to supply a suitable initial timestep/load-factor through the NSUB or DELTIM command for convergence and overall efficiency.

LOAD STEP OPTIONS

```
LOAD STEP NUMBER. . . . . 1
TIME AT END OF THE LOAD STEP. . . . . 1.0000
AUTOMATIC TIME STEPPING . . . . . ON
  INITIAL NUMBER OF SUBSTEPS . . . . . 1
  MAXIMUM NUMBER OF SUBSTEPS . . . . . 1000
  MINIMUM NUMBER OF SUBSTEPS . . . . . 1
  START WITH TIME STEP FROM PREVIOUS SUBSTEP . YES
MAXIMUM NUMBER OF EQUILIBRIUM ITERATIONS. . . . 15
STEP CHANGE BOUNDARY CONDITIONS . . . . . NO
TERMINATE ANALYSIS IF NOT CONVERGED . . . . .YES (EXIT)
CONVERGENCE CONTROLS. . . . .USE DEFAULTS
COPY INTEGRATION POINT VALUES TO NODE . . . . .YES, FOR ELEMENTS WITH
  ACTIVE MAT. NONLINEARITIES
PRINT OUTPUT CONTROLS . . . . .NO PRINTOUT
DATABASE OUTPUT CONTROLS. . . . .ALL DATA WRITTEN FOR THE LAST SUBSTEP
```

SOLUTION MONITORING INFO IS WRITTEN TO FILE= file.mntr

```
DISP CONVERGENCE VALUE = 0.2416E-01 CRITERION= 0.2866E-04
EQUIL ITER 1 COMPLETED. NEW TRIANG MATRIX. MAX DOF INC= 0.5732E-03
FORCE CONVERGENCE VALUE = 0.2847 CRITERION= 0.2070E-01
curEqn= 5606 totEqn= 5605 Job CP sec= 89.266
Factor Done= 100% Factor Wall sec= 0.008 rate= 382.4 Mflops
DISP CONVERGENCE VALUE = 0.1777E-04 CRITERION= 0.2870E-04 <<< CONVERGED
EQUIL ITER 2 COMPLETED. NEW TRIANG MATRIX. MAX DOF INC= 0.9573E-06
FORCE CONVERGENCE VALUE = 0.4785E-02 CRITERION= 0.2113E-01 <<< CONVERGED
>>> SOLUTION CONVERGED AFTER EQUILIBRIUM ITERATION 2
```

Step 5 .The results of the analysis are shown in above, **convergence was achieved** at the factored load condition; therefore the vessel passes Elastic-Plastic Analysis for this load case.

5.0 Concluding Remarks

Level 1 assessment = Failed to meet criterion

Level 2 assessment = Failed to meet criterion

Level 3 assessment (Finite Element Analysis)

1. Elastic Stress Analysis and Local Failure = **Pass**

2. Limit load Analysis = **Pass**

3. Elastic-Plastic Analysis = **Pass**

The alternative methods of Limit load analysis and Elastic-Plastic analysis are used to verify the results obtained from elastic stress analysis, thus the shell with corroded portion found much safer for its continual operation. The vessel is exempted from Re-rate, Repair and Retirement.

References

ANSYS (version 15.0) [Computer software]

ASME Boiler & Pressure Vessel Code, Section VIII Div.2, part 5 Ed 2017

Dennis R. Moss, Pressure Vessel Design Manual Ed 4

Fitness-For-Service API 579-1/ASME FFS-1, Ed 2016

Fitness-For-Service Manual API 579-2/ASME FFS-2 2009

ASME Section II Part D Edition 2017.